

Design and Analysis of Hydraulic Forklift Using Foot Operated

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ABSTRACT

The project focused on design and analysis of hydraulic forklift using foot operated. The calculated design and mass properties of parts and subassemblies according to the material selection (mild steel) to ensure the stability of the forklift, which is capable of lifting the fork up to 300 kg and lift up to an height of 250 mm. Then the stress analysis done on the important parts and subassemblies using finite element analysis method (FEA). Results show that the new design is safe to use under working conditions

Keywords: Permissible stress Bending moment Bulking Hardness

Nomenclature

The nomenclature should be in alphabetical order with Greek symbol, also in alphabetical order. Subscripts and superscripts should follow Greek symbols and should be identified with separate headings. Nomenclature entries should have the units identified. For example:

Q	Flow rate
A, A0	Area, maximum area of the exit cross section
g	Gravitational acceleration
H	Head behind the valve
He	Head drop before the minimum height section
Zc	Height for the minimum section
Cd	Discharge coefficient
H _{max} , H _{normal} , H _{min}	Maximum, normal and minimum head
K	Turbulent kinetic energy
ε	Turbulent dissipation energy

I. INTRODUCTION

Hydraulics are used in forklifts to lift the load-bearing prongs up off the ground and hold the load in the air while the forklift moves. The hydraulic system in a forklift has been described as

the heart of the vehicle, and the hydraulic lifting system does most of the work and without it, the vehicle won't be able to move pallets

In general the forklift can be defined as a tool capable of lifting hundreds of kilograms. A forklift is a vehicle similar to a small truck that has two metal forks on the front used to lift cargo. The forklift operator drives the forklift forward until the forks push under the cargo, and can then lift the cargo several feet in the air by operating the forks.

The forks, also known as blades or tines, are usually made out of steel and can lift up to a few tons. Forklifts are either powered by gasoline, propane, or electricity. Electric forklifts rely on batteries to operate. Gasoline or propane forklifts are sometimes stronger or faster than electric forklifts, but they are more difficult to maintain, and fuel can be costly. Electric forklifts and hydraulic forklift are great for warehouse use because they do not give off noxious fumes like gas powered machines do.

A forklift is a one type of power industrial truck that comes in different shapes, sizes and forms. A forklift can be called a pallet truck, rider truck, fork truck or lift truck. Yet, the ultimate purpose of forklift is the same to safely allow one person to lift and move large heavy loads with little effort. Hydraulic forklift also known as hydraulic hand pallet is a tool used to lift and transport heavy load for long distances with the help of pallet.

Pallet jacks are the most compact and modern form of forklift and are intended to move heavy and light weight material within a warehouses. For the purpose of training, a forklift is a small or large industrial truck with power operated platform. Like other forms of forklift hydraulic forklift doesn't require any kind of electric power source or diesel and gasoline because hydraulic forklift works on principle of hydrostatic force transmission.

Lifting of heavy loads is accomplished with the help of hydraulic cylinder in the forklift. Cylinder is generally fitted at lower parts of fork. Forklifts are most often used in warehouses, but some are meant to be used outdoors. The vast

majority of rough terrain forklifts operate on gasoline, but some usediesel or natural gas. Rough terrain forklifts have the highest lifting capacity of all forklifts and heavy duty tires (like those found on trucks), making it possible to drive them on uneven surfaces outdoors. Forklifts have revolutionized warehouse work. They made it possible for one person to move thousands of pounds at once. Well-maintained and safely operated forklifts make lifting and transporting cargo

II. MATERIALS AND METHOD

Due to its excellent properties, mild steel has become an in-demand material in various industries. It has unparalleled weld ability and machinability, which has led to an exponential increase in its usage. In this article, we will discuss the importance of mild steel, its uses and how it's made. **Mild steel is a type of low carbon steel.** Carbon steels are metals that contain a small percentage of carbon (max 2.1%) which enhances the properties of pure iron. The carbon content varies depending on the requirements for the steel. Low carbon steels **contain carbon in the range of 0.05 to 0.25 percent.**

Table 1. Physical properties of mild steel

Properties	Carbon Steels	Alloy Steels	Stainless Steels	Tool Steels
Density (2000 kg/m ³)	785	785	775-81	772-8.0
Elastic Modulus (GPa)	190-210	190-210	190-210	190-210
Poisson's Ratio	0.27-0.3	0.27-0.3	0.27-0.3	0.27-0.3
Thermal Expansion (10 ⁻⁶ /K)	11-15.6	10-16	10-20.7	14-15.1
Melting Point (°C)			1571-1654	
Thermal Conductivity (W/m-K)	24.3-66.2	26-48.6	11.2-36.7	16.9-48.3
Specific Heat (J/kg-K)	430-2081	452-1499	430-500	
Electrical Resistivity (10 ⁻⁸ Ω-m)	130-1250	210-1251	75.7-1020	
Tensile Strength (MPa)	276-1862	758-1882	516-827	640-2000
Yield Strength (MPa)	166-758	366-1713	237-552	380-440
Percent Elongation (%)	10-32	4-31	12-40	5-25
Hardness (Brinell 3000kg)	91-388	149-627	137-595	210-620

Chart: The Balance - Source: [alibaba](#)

The above considerations are validated for further proceedings and from the Table 2. There are different grades of mild steel. But they all have carbon content within the above-mentioned limits. Other elements are added to improve useful properties like corrosion resistance, wear resistance and tensile

strength

Table 1. Chemical composition of mild steel

CHEMICAL COMPOSITION (%)				
Fe	Mn	S	P	C
98.81-99.26	0.6-0.9	0.05	0.04	0.14-0.2

PHYSICAL PROPERTIES

YIELD STRENGTH	TENSILE STRENGTH	THERMAL CONDUCTIVITY	MELTING POINT	HARDNESS	SPECIFIC HEAT CAPACITY
275	475	51.9	1523	143	0.472

Design Of Fork Design Of Weld (FORK)

- 1) F.O.S = 4.7
- 2) MS Yield Point Stress (σ_y) = 300N/mm²
- 3) Load Applied = 300KG = 2943N
- 4) $b=5d$ (standard material ratio)

(a) Allowable (or) Permissible Stress σ_b (or) $\sigma_t = \sigma_y / F.O.S$ (n)
 $= 300 / 4.7$
 $= 63.82$
 $= 64 \text{ N/m}^2$

(b) Maximum Bending Moment M_t $M_b = -(300 \times 600)$
 $= -180 \times 10^3 \text{ N-mm}$

(c) $Z = M_b / \sigma$
 $= 180 \times 10^3 / 64$
 $Z = 22812.5$

(d) $Z = bd^2/6$ [$b=5d$]
 $22812.5 = 5d \times d^2 / 6$
 $22812.5 = 5d^3 / 6$
 $22812.5 = 0.83d^3$
 $3388.5 = d^3$ $(3388.5)^{1/3} = dd = 15.02$

(e) $b = 5d$ $b = 5 \times 15.02$
 $b = 75.10$

Maximum Load with the Plate

$P = \text{Area} \times \text{stress}$
 $\text{Area} = \text{Breath} \times \text{Thickness}$

$$\begin{aligned} &= B \times T \times \sigma t \\ &= 75 \times 15 \times 64 \\ &= 72 \times 10^3 \\ \text{Maximum Load} &= 72 \times 10^3 \end{aligned}$$

Case (i)

Uniform Distribution Load

$$\begin{aligned} (1) \text{ Moment Of Inertia } I_1 &= I_2 = bd^3/12 \\ &= 75 \times 15^3/12 \\ &= 21093.75 \text{ mm}^4 \end{aligned}$$

$$\begin{aligned} (2) \text{ Find } W/\text{mm} &= 3000/600 \\ &= 5 \text{ N/mm} \end{aligned}$$

$$\begin{aligned} (3) \text{ Bending Moment (M.A)} &= (-w \times l^2)/2 \\ &= -5 \times 600^2/2 \\ &= -900 \times 10^3 \text{ N-mm.} \end{aligned}$$

$$\begin{aligned} (4) \text{ Deflection (y}_{\text{max}}) &= W \times L^4/8EI \\ &= 5 \times 600^4/8 \times 200 \times 10^3 \times 21093.75 \\ Y_{\text{max}} &= 19.2 \text{ mm} \end{aligned}$$

$$\begin{aligned} \text{To The Length Of Weld} & \\ &= 140 + 74 + 140 \\ &= 214 \text{ mm} \end{aligned}$$

$$\begin{aligned} (1) \text{ Direct shear load per unit length of weld.} & \\ Pd &= p/l \\ &= 2943/214 \\ &= 13.75 \text{ N/mm} \end{aligned}$$

$$\begin{aligned} (2) \text{ Load due to bending per unit length of weld.} & \\ Pn &= p \times e/Z_w, e = 800. \\ Z_{\text{wtop}} &= (Zbd + d^2)/3. \\ &= (2 \times 140 \times 74 + 74^2)/3 \\ &= 8732 \text{ mm}^2. \\ Z_{\text{wbottom}} &= (d^2 \times (2b + d))/3(b + d). \\ &= (74^2 \times (2 \times 140 + 74))/3(140 + 75). \\ &= 3019.47 \text{ N/mm}^2. \end{aligned}$$

$$\begin{aligned} \text{Select the smaller value as the permissible value} & \\ Z_w &= 3019.47 \text{ N/mm}^2. \\ Pn &= (p \times e)/Z_w. \\ &= (2943 \times 800)/3019.4 \\ Pn &= 780 \text{ N/mm.} \end{aligned}$$

$$\begin{aligned} (3) \text{ Resultant Load } Pr &= \sqrt{(Pd^2 + Pn^2)} \\ &= \sqrt{(13.75^2 + 780^2)} \\ &= 780 \text{ N/mm.} \end{aligned}$$

$$\begin{aligned} (4.) \text{ Size of Weld.} & \\ \text{Allowable stress} &= (P \times r)/0.707w \times 13677 \\ &= 780 / (0.707 \times w \times 1) \\ W &= 3 \text{ mm} \end{aligned}$$

Design of Buckling Consideration in C section frame(Guide column) with eccentric load.

- Outer face height (D)=76mm
 - Outer face Width (B)=40mm.
 - Thickness (t)=5mm.
 - Length of C frame =100cm=1000mm.
 - Material =Mild Steel
 - E for MS=200G
- $$Pa = 200 \times 10^3 \text{ N/mm}^2.$$

$$\begin{aligned} (1) \text{ Cross Section Area (A).} &= (D \times B) - (d \times b) \\ &= (76 \times 40) - (66 \times 35) A = 730 \text{ mm}^2. \end{aligned}$$

$$\begin{aligned} (2) \text{ Moment of Inertia (I).} & \\ I &= ((B \times D^3)/12) - (b \times d^3)/12) \\ &= ((40 \times 76^3)/12) - (35 \times 66^3/12)) \\ &= (1499834.667 - 838530) I = 661.304 \times 10^3 \text{ mm}^4. \end{aligned}$$

$$\begin{aligned} (3) \text{ Find } y^- & \\ \text{Thickness of C frame (t)} &= 5 \text{ mm.} \\ y^- &= t + (d/2) \\ &= 5 + d/2. \\ y^- &= 38 \text{ mm.} \end{aligned}$$

$$\begin{aligned} (4) \text{ Section Modulus (Z).} &= I/y^- \\ &= (661.304 \times 10^3)/38. \\ Z &= 17402.73 \text{ mm}^3. \end{aligned}$$

$$\begin{aligned} (5.) \text{ Equivalent Length (Le) Length of C} & \\ \text{frame} &= 1000 \text{ mm.} \\ Le &= L/\sqrt{2}. \\ &= 1000/\sqrt{2}. \\ Le &= 707.106 \text{ mm.} \end{aligned}$$

$$\begin{aligned} (6.) \text{ Maximum B.M. } M_{\text{max}} &= P \times e \times \sec (Le/2) \sqrt{(P/EI)}. \\ &= 3000 \times 600 \times \sec ((707.106/2) \sqrt{(3000/200 \times 10^3 \times 661.304 \times 10^3)}) \\ M_{\text{max}} &= 2.731 \times 10^6 \text{ N/mm.} \end{aligned}$$

III. RESULTS AND DISCUSSION

- The existing forklift design has its limitation in lifting a fork using lever which is connected to pump but we have made some modification in which it can be byfoot operated.
- The pedal operated forklift which have been design to load capacity of 3000 N
- The weight reduction of the structure reduced to 30 percent

FROM DESIGN CALCULATION:

- Permissible Stress for fork is 63.82 = 64 N/m²

- Maximum Load with the Plate is Maximum Load = 72×10^3 N.

IN CASE OF UNIFORM DISTRIBUTED LOAD:
Deflection is $Y_{max} = 19.2$ mm
Consideration in C section frame (Guide column)

□ Cross Section Area is 730mm.

- Equivalent Length (Le) is = 707.106mm.

- Maximum B.M is = 2.731×10^6 N/mm.

- So by comparing the Design calculations values in above points is equal to the values of the Ansys report while Applying the same material.

IV. CONCLUSION

We conclude that, this project will be helpful for small scale industries as it is easy to operate with less cost and indirectly it will save the labor cost.

Savings resulting from the use of this machine will make it pay for itself within a short period of time and it can be a great companion in any field dealing with rusted and unused metals.

It is a mechanical device, does not require electricity as well as any external source of battery. The development of mechanical forklift assures the ergonomic comfort to the operator or worker and reduces time required for manual lifting and handling.

This increases efficiency of productivity and it provides safety of operator while handling of the material. It lifts maximum load up to 400 kg at maximum height of 300 mm.

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